

Proper Matching of Bolt & Nut Strengths is Critical

by Joe Greenslade



It is obvious that if a bolted joint is subjected to enough force, sooner or later, something must fail. From a design point of view, some types of failures are more desirable than others. One widely accepted principle of joint design is as follows:

The desired mode of failure of a bolted joint is the complete fracturing of the bolt or screw instead of the stripping of the thread in the nut or in the internally threaded component.

The reasoning behind this principle is simple. When a bolt or screw breaks during initial assembly, the joint failure is obvious. When this happens, the broken bolt or screw can be replaced before more serious damage occurs in the assembly.

When an internal thread strips during assembly, the joint failure is frequently not obvious by visual inspection. This can create very dangerous situations in which the assembly may completely come apart and fail when the assembly begins its function and is subjected to its operating loads. These types of failures can result in huge financial losses and can possibly put lives in danger.

Examples of correct nut selections for given bolts:

Thread Size	Bolt Grade Or Property Class	Bolt's Ultimate Tensile Strength	Nut Grade Or Property Class	Nut Proof Load Capacity
3/4-10	SAE Grade 5	40,100 lbs.	SAE Grade 5 Or ASTM A563 Grade B	40,100 lbs.
M16X2.0	Property Class 10.9	163,000 N	Property Class 10	164,900 N

The following rule should be followed when nuts are selected for use with bolts or screws:

Whenever possible, a nut should have a "proof load capacity" equal to or greater than the minimum "ultimate tensile strength" of the bolt or screw with which it will be used.

The proof load capacity of nuts and the ultimate tensile strength of screws and bolts can be found in the various fastener standards including those published by the American Society of Testing and Materials (ASTM), the Society of Automotive Engineers (SAE), and the International Standards Organization (ISO). The specific standards used as references for this article are as follows:

- SAE J449
- SAE J995
- ISO 898-1
- ISO 898-2
- ASTM A193
- ASTM A194
- ASTM A307
- ASTM A325
- ASTM A354
- ASTM A449
- ASTM A490

A great general reference containing data from all of these sources and more is "Every Thing An Engineer Should Know About Threaded Fasteners," written by Alexander Blake. This book is published by Marcel Dekker, Inc. and is available from the Industrial Fastener Institute (IFI) at 216-241-1482.

Below are charts taken from the above mentioned standards for inch and metric coarse thread bolt and nut sizes covering 1/4 inch through 1 inch and M5 through M24. These charts show the minimum ultimate strength values for the bolts and the proof load capacity values for the nuts.



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Mr. Greenslade holds twelve U.S. patents on various fastener related products. He has authored over 136 trade journal articles on fastener applications, manufacturing and quality issues. He is one of the fastener industry's most frequent speakers at trade association meetings and conferences. He is the youngest person ever inducted to the Fastener Industry Hall of Fame.

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In addition to guiding the activities of Greenslade & Company, Mr. Greenslade works as a consultant with fastener suppliers and end users on product design, applications engineering, and quality issues. In this capacity he works to resolve fastener applications problems, to help select the best fastening approaches in new product designs, to assist in the standardization of fasteners used within an organization, and to provide training on various aspects of fastening technology and fastener quality assurance. He also serves as Expert Witness in litigation involving fastener related issues.

METRIC BOLT ULTIMATE TENSILE STRENGTH BY ISO PROPERTY CLASS (NEWTONS)

Size	3.6	4.6	4.8	5.8	6.8	8.8	9.8	10.9	12.9
M5X0.08	4,690	5,680	5,960	7,100	8,520	11,350	12,800	14,800	17,300
M6X1.0	6,630	8,040	8,440	10,000	12,100	16,100	18,100	20,900	24,500
M8X1.25	12,100	14,600	15,400	18,300	22,000	29,200	32,900	38,100	44,600
M10X1.5	19,100	23,200	24,400	29,000	34,800	46,400	52,200	60,300	70,800
M12X1.75	27,800	33,700	35,400	42,200	50,600	67,400	75,900	87,700	103,000
M14X2.0	38,000	46,000	48,300	57,500	69,000	92,000	104,000	120,000	140,000
M16X2.0	51,800	62,800	65,900	78,500	94,000	125,000	141,000	163,000	192,000
M18X2.5	63,400	76,800	80,600	96,000	115,000	159,000	NA	200,000	234,000
M20X2.5	80,800	98,000	103,000	122,000	147,000	203,000	NA	255,000	299,000
M22X2.5	100,000	121,000	127,000	152,000	182,000	252,000	NA	315,000	370,000
M24X3.0	116,000	141,000	148,000	176,000	212,000	293,000	NA	367,000	431,000

METRIC NUT PROOF LOAD CAPACITY BY ISO PROPERTY CLASS (NEWTONS)

Size	04	05	4	5	6	8	9	10	12		
			Style 1	Style 1	Style 1	Style 1	Style 2	Style 2	Style 1	Style 1	Style 2
M5X0.08	5,400	7,100	NA	8,250	9,500	12,140	NA	13,000	14,800	16,200	16,300
M6X1.0	7,640	10,000	NA	11,700	13,500	17,200	NA	18,400	20,900	22,900	23,100
M8X1.25	13,900	18,300	NA	21,600	24,900	31,800	NA	34,400	38,100	41,700	42,500
M10X1.5	22,000	29,000	NA	34,200	39,400	50,500	NA	54,500	60,300	66,100	67,300
M12X1.75	32,000	42,200	NA	51,400	59,000	74,200	NA	80,100	88,500	98,600	100,300
M14X2.0	43,700	57,500	NA	70,200	80,500	101,200	NA	109,300	120,800	134,600	136,900
M16X2.0	59,700	78,500	NA	95,800	109,900	138,200	NA	149,200	164,900	183,700	186,800
M18X2.5	73,000	96,000	97,900	121,000	138,200	176,600	170,900	176,600	203,500	NA	230,400
M20X2.5	93,100	122,000	125,000	154,400	176,400	225,400	218,100	225,400	259,700	NA	294,000
M22X2.5	115,100	151,500	154,500	190,900	218,200	278,800	269,700	278,800	321,200	NA	363,600
M24X3.0	134,100	176,500	180,000	222,400	254,200	324,800	314,200	324,800	374,200	NA	423,600

- * (1) The values in this chart are for Hex and Heavy Hex Nuts with Class 6H threads. Overtapped nuts have proof load capacities 75% of those shown above.
- (2) The values in the chart above do not apply to Jam Nuts or Slotted Nuts. Their values are lower due to less thread engagement than that of Hex and Heavy Hex Nuts.

INCH NUT PROOF LOAD CAPACITY (POUNDS FORCE)

Size	Hex Nuts*			Heavy/Thick Hex Nuts*			
	SAE Grade 2	SAE Grade 5	SAE Grade 8	ASTM A563 Grade A	ASTM A563 Grade C	ASTM A563 Grade D	ASTM A563 Grade DH Grade DH3
	ASTM A563 Grade A	ASTM A563 Grade B	ASTM A563 Grade DH		ASTM Grade C3	ASTM A194 Grade 2	ASTM A194 Grade 2H
1/4-20	2,850	3,800	4,750	3,180	4,229	5,247	5,565
5/16-18	4,700	6,300	7,850	5,240	6,969	8,646	9,170
3/8-16	7,000	9,300	11,600	7,750	10,308	12,788	13,563
7/16-14	9,550	12,800	15,900	10,600	14,098	17,490	18,550
1/2-13	12,800	17,000	21,300	14,200	18,886	23,430	24,850
9/16-12	16,400	21,800	27,300	18,200	24,206	30,030	31,850
5/8-11	20,300	27,100	33,900	22,600	30,058	37,290	39,550
3/4-10	30,100	40,100	50,100	33,400	44,422	55,110	58,450
7/8-9	41,600	55,400	69,300	46,200	61,446	76,230	80,850
1-8	54,500	72,700	90,900	60,600	80,598	99,990	106,050

- * (1) The values in this chart are for Hex and Heavy Hex Nuts with Class 2B threads. Overtapped nuts have proof load capacities 75% of those shown above.
- (2) The values in the chart above do not apply to Jam Nuts or Slotted Nuts. Their values are lower due to less thread engagement than that of Hex and Heavy Hex Nuts.

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PROPER MATCHING

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INCH BOLT ULTIMATE TENSILE STRENGTH BY GRADE (POUNDS FORCE)					
Size	ASTM A307	SAE Grade 2	SAE Grade 5 ASTM A449 A325	ASTM A193 Grade B7	SAE Grade 8 ASTM A490
1/4-20	1,910	2,350	3,800	3,975	4,750
5/16-18	3,140	3,880	6,300	6,550	7,850
3/8-16	4,650	5,740	9,300	9,688	11,600
7/16-14	6,380	7,870	12,800	13,250	15,900
1/2-13	8,510	10,500	17,000	17,750	21,300
9/16-12	10,900	13,500	21,800	22,750	27,300
5/8-11	13,600	16,700	27,100	28,250	33,900
3/4-10	20,000	24,700	40,100	41,750	50,100
7/8-9	27,700	NA	55,400	57,750	69,300
1-8	36,400	NA	72,700	75,750	90,900

When determining what nut grade to use with a given bolt:

- Find the bolt's ultimate tensile strength by thread size and strength grade in the appropriate chart.
- Locate the thread size line in the appropriate nut chart that matches the bolt's thread size and select the nut grade to use where the nut's proof load capacity value is equal to or greater than the bolt's ultimate tensile strength.

Fastener users should always make the final choice of which nut to use with which bolt, but if asked, suppliers should share these principles with them. Fastener users should be warned that using a lower grade nut to save on the initial purchase price of fasteners could prove to be false economy.

If the nut's threads strip during installation and the failure goes undetected, the entire assembly may fail when it is put into use. When this type of catastrophic failure occurs the supplier usually has to defend himself against accusations about having supplied fasteners of poor quality. This is unfair and unjust when the root cause of the failure is poor nut selection by the user. ■

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